

Galerkin Finite Element Model for Heat Transfer

Introductory Course on Multiphysics Modelling

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2 Local differential formulation

- Partial Differential Equation
- Initial and boundary conditions
- Initial-Boundary-Value Problem

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- Test functions
- Weighted formulation and weak variational form

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- Approximation
- Transient heat transfer (ordinary differential equations)
- Stationary heat transfer (algebraic equations)

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Notation remarks

- The **index notation** is used with summation over the index i .
- Consequently, the **summation rule** is also applied for the approximation expressions, that is, over the indices $r, s = 1, \dots, N$ (where N is the number of degrees of freedom).
- The symbol $(\dots)_{|i}$ means a (generalized) **invariant partial differentiation** over the i -th coordinate:

$$(\dots)_{|i} = \frac{\partial(\dots)}{\partial x_i}.$$

The invariance involves the so-called Christoffel symbols (in the case of curvilinear systems of reference).

- Symbols dV and dS are completely omitted in all the integrals presented below since it is obvious that one integrates over the specified domain or boundary. Therefore, one should understand that:

$$\int_{\mathcal{B}} (\dots) = \int_{\mathcal{B}} (\dots) dV(\mathbf{x}), \quad \int_{\partial\mathcal{B}} (\dots) = \int_{\partial\mathcal{B}} (\dots) dS(\mathbf{x}).$$

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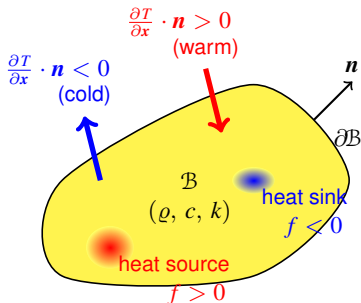
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PDE for Heat Transfer Problem



■ Material data:

$\varrho = \varrho(\mathbf{x})$ – the density $\left[\frac{\text{kg}}{\text{m}^3}\right]$

$c = c(\mathbf{x})$ – the thermal capacity $\left[\frac{\text{J}}{\text{kg} \cdot \text{K}}\right]$

$k = k(\mathbf{x})$ – the thermal conductivity $\left[\frac{\text{W}}{\text{m} \cdot \text{K}}\right]$

■ Known fields:

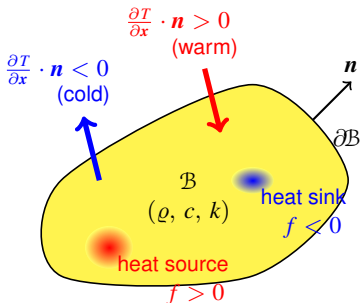
$f = f(\mathbf{x}, t)$ – the heat production rate $\left[\frac{\text{W}}{\text{m}^3}\right]$

$u_i = u_i(\mathbf{x}, t)$ – the convective velocity $\left[\frac{\text{m}}{\text{s}}\right]$

■ The unknown field:

$T = T(\mathbf{x}, t) = ?$ – the temperature $[\text{K}]$

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■ The unknown field:

$T = T(\mathbf{x}, t) = ?$ – the temperature $[\text{K}]$

Heat transfer equation

$\rho c \dot{T} + q_{i|i} - f = 0$ where the heat flux vector $\left[\frac{\text{W}}{\text{K}}\right]$:

$$q_i = q_i(T) = \begin{cases} -k T_{|i} & \text{– for conduction (only),} \\ -k T_{|i} + \rho c u_i T & \text{– for conduction and convection,} \end{cases}$$

and $\dot{T} = \frac{\partial T}{\partial t}$ is the time rate of change of temperature $\left[\frac{\text{K}}{\text{s}}\right]$.

Initial and boundary conditions

The initial condition (at $t = t_0$)

- $T(\mathbf{x}, t_0) = T_0(\mathbf{x})$ in \mathcal{B}

Prescribed field:

$T_0 = T_0(\mathbf{x})$ – the initial temperature [K]

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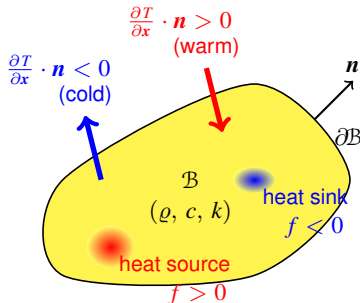
The boundary conditions (on $\partial\mathcal{B}$)

- the Dirichlet type:
 $T(\mathbf{x}, t) = \hat{T}(\mathbf{x}, t)$ on $\partial\mathcal{B}_T$
- the Neumann type:
 $-q_i(T(\mathbf{x}, t)) n_i = \hat{q}(\mathbf{x}, t)$ on $\partial\mathcal{B}_q$

Prescribed fields:

$\hat{T} = \hat{T}(\mathbf{x}, t)$ – the temperature [K]

$\hat{q} = \hat{q}(\mathbf{x}, t)$ – the inward heat flux $\left[\frac{\text{W}}{\text{m}^2}\right]$



Initial-Boundary-Value Problem

IBVP of the heat transfer

Find $T = T(\mathbf{x}, t)$ for $\mathbf{x} \in \mathcal{B}$ and $t \in [t_0, t_1]$ satisfying the **equation of heat transfer** by conduction (a), or by conduction and **convection** (b):

$$\varrho c \dot{T} + q_{i|i} - f = 0 \quad \text{where} \quad q_i = q_i(T) = \begin{cases} -k T_{|i} & \leftarrow \text{(a)} \\ -k T_{|i} + \varrho c u_i T & \leftarrow \text{(b)} \end{cases}$$

with the **initial condition** (at $t = t_0$):

$$T(\mathbf{x}, t_0) = T_0(\mathbf{x}) \quad \text{in } \mathcal{B},$$

and subject to the **boundary conditions**:

$$T(\mathbf{x}, t) = \hat{T}(\mathbf{x}, t) \quad \text{on } \partial\mathcal{B}_T, \quad -q_i(T(\mathbf{x}, t)) n_i = \hat{q}(\mathbf{x}, t) \quad \text{on } \partial\mathcal{B}_q,$$

where $\partial\mathcal{B}_T \cup \partial\mathcal{B}_q = \partial\mathcal{B}$ and $\partial\mathcal{B}_T \cap \partial\mathcal{B}_q = \emptyset$.

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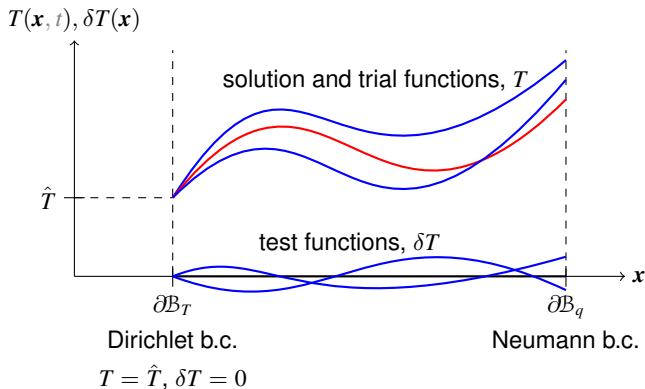
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Test functions



Test function $\delta T(x)$ is an arbitrary (but sufficiently regular) function defined in \mathcal{B} , which meets the *admissibility condition*:

$$\delta T = 0 \quad \text{on } \partial B_T.$$

Notice that **test functions are always time-independent**.

Weighted formulation and weak variational form

Weighted integral formulation

$$\int_{\mathcal{B}} \left(\varrho c \dot{T} + q_{i|i} - f \right) \delta T = 0 \quad (\text{for every } \delta T)$$

Weighted formulation and weak variational form

Weighted integral formulation

$$\int_{\mathcal{B}} \varrho c \dot{T} \delta T + \int_{\mathcal{B}} q_{i|i} \delta T - \int_{\mathcal{B}} f \delta T = 0 \quad (\text{for every } \delta T)$$

The term $q_{i|i}$ introduces the second derivative of T : $q_{i|i} = -k T_{|ii} + \dots$.
However, the heat PDE needs to be satisfied in the integral sense.
Therefore, the requirements for T can be **weaken** as follows.

Weighted formulation and weak variational form

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- Integrating by parts (using the divergence theorem)

$$\int_{\mathcal{B}} q_{i|i} \delta T = \int_{\mathcal{B}} (q_i \delta T)_{|i} - \int_{\mathcal{B}} q_i \delta T_{|i} = \int_{\partial \mathcal{B}} q_i \delta T n_i - \int_{\mathcal{B}} q_i \delta T_{|i}$$

Weighted formulation and weak variational form

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- Using the **Neumann b.c** and the **property of test function**

$$\int_{\partial \mathcal{B}} q_i n_i \delta T = \int_{\partial \mathcal{B}_q} \underbrace{q_i n_i}_{-\hat{q}} \delta T + \int_{\partial \mathcal{B}_T} q_i n_i \underbrace{\delta T}_0 = - \int_{\partial \mathcal{B}_q} \hat{q} \delta T$$

Weighted formulation and weak variational form

Weighted integral formulation

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After integrating by parts and using the Neumann boundary condition

$$\int_{\mathcal{B}} q_{i|i} \delta T = - \int_{\partial \mathcal{B}_q} \hat{q} \delta T - \int_{\mathcal{B}} q_i \delta T_{|i}$$

Weak variational form

$$\int_{\mathcal{B}} \varrho c \dot{T} \delta T - \int_{\mathcal{B}} q_i \delta T_{|i} - \int_{\partial \mathcal{B}_q} \hat{q} \delta T - \int_{\mathcal{B}} f \delta T = 0 \quad (\text{for every } \delta T)$$

Weighted formulation and weak variational form

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Weak variational form

$$\int_{\mathcal{B}} \rho c \dot{T} \delta T - \int_{\mathcal{B}} q_i \delta T_{|i} - \int_{\partial \mathcal{B}_q} \hat{q} \delta T - \int_{\mathcal{B}} f \delta T = 0 \quad (\text{for every } \delta T)$$

Now, only the first order spatial-differentiability of T is required.

In this formulation the Neumann boundary condition is already met (it has been used in a *natural* way). Therefore, the only additional requirements are the Dirichlet boundary condition and the initial condition.

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Approximation functions and space

The spatial approximation of solution in the domain \mathcal{B} is accomplished by a linear combination of (global) **shape functions**,
 $\phi_s = \phi_s(\mathbf{x})$,

$$T(\mathbf{x}, t) = \theta_s(t) \phi_s(\mathbf{x}) \quad (s = 1, \dots, N; \text{ summation over } s)$$

where $\theta_s(t)$ [K] are (time-dependent) coefficients – the **degrees of freedom** (N is the total number of degrees of freedom). Consistent result is obtained now for the time rate of temperature

$$\dot{T}(\mathbf{x}, t) = \dot{\theta}_s(t) \phi_s(\mathbf{x}) \quad \text{where} \quad \dot{\theta}_s(t) = \frac{d\theta_s(t)}{dt} \quad \left[\frac{\text{K}}{\text{s}} \right].$$

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Distinctive feature of the Galerkin method:

The same shape functions are used to approximate the solution as well as the test function, namely

$$\delta T(\mathbf{x}) = \delta \theta_r \phi_r(\mathbf{x}) \quad (r = 1, \dots, N; \text{ summation over } r).$$

Transient heat transfer (system of ODEs)

To reduce the “regularity” requirements for solution the approximations

$$T = \theta_s \phi_s \quad \left(\dot{T} = \dot{\theta}_s \phi_s \right), \quad \delta T = \delta \theta_r \phi_r$$

are used for the **weak variational form** of the heat transfer problem

$$\int_{\mathcal{B}} \varrho c \dot{T} \delta T - \int_{\mathcal{B}} q_i \delta T_{|i} - \int_{\partial \mathcal{B}_q} \hat{q} \delta T - \int_{\mathcal{B}} f \delta T = 0.$$

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$$\boxed{1} \quad \int_{\mathcal{B}} \varrho c \dot{T} \delta T = \dot{\theta}_s \delta \theta_r \int_{\mathcal{B}} \varrho c \phi_s \phi_r = \dot{\theta}_s \delta \theta_r M_{rs}$$

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$$\mathbf{1} \quad \int_{\mathcal{B}} \varrho c \dot{T} \delta T = \dot{\theta}_s \delta \theta_r \int_{\mathcal{B}} \varrho c \phi_s \phi_r = \dot{\theta}_s \delta \theta_r \mathbf{M}_{rs}$$

$$\begin{aligned} \mathbf{2} \quad \int_{\mathcal{B}} q_i \delta T_{|i} &= \int_{\mathcal{B}} (-k T_{|i} + \varrho c u_i T) \delta T_{|i} = \theta_s \delta \theta_r \int_{\mathcal{B}} (-k \phi_{s|i} + \varrho c u_i \phi_s) \phi_{r|i} \\ &= \theta_s \delta \theta_r \mathbf{K}_{rs} \end{aligned}$$

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$$\mathbf{3} \quad \int_{\partial \mathcal{B}_q} \hat{q} \delta T = \delta \theta_r \int_{\partial \mathcal{B}_q} \hat{q} \phi_r = \delta \theta_r \mathbf{Q}_r$$

Transient heat transfer (system of ODEs)

To reduce the “regularity” requirements for solution the approximations

$$T = \theta_s \phi_s \quad \left(\dot{T} = \dot{\theta}_s \phi_s \right), \quad \delta T = \delta \theta_r \phi_r$$

are used for the **weak variational form** of the heat transfer problem

$$\int_{\mathcal{B}} \varrho c \dot{T} \delta T - \int_{\mathcal{B}} q_i \delta T_{|i} - \int_{\partial \mathcal{B}_q} \hat{q} \delta T - \int_{\mathcal{B}} f \delta T = 0.$$

$$\textbf{1} \quad \int_{\mathcal{B}} \varrho c \dot{T} \delta T = \dot{\theta}_s \delta \theta_r \int_{\mathcal{B}} \varrho c \phi_s \phi_r = \dot{\theta}_s \delta \theta_r \mathbf{M}_{rs}$$

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$$\textbf{3} \quad \int_{\partial \mathcal{B}_q} \hat{q} \delta T = \delta \theta_r \int_{\partial \mathcal{B}_q} \hat{q} \phi_r = \delta \theta_r \mathbf{Q}_r$$

$$\textbf{4} \quad \int_{\mathcal{B}} f \delta T = \delta \theta_r \int_{\mathcal{B}} f \phi_r = \delta \theta_r \mathbf{F}_r$$

Transient heat transfer (system of ODEs)

Matrix formulation of the heat transfer problem

$$\left[M_{rs} \dot{\theta}_s - K_{rs} \theta_s - (Q_r + F_r) \right] \delta \theta_r = 0 \quad \text{for every } \delta \theta_r.$$

This produces the following system of first-order **ordinary differential equations** (for $\theta_s = \theta_s(t) = ?$):

$$\boxed{M_{rs} \dot{\theta}_s - K_{rs} \theta_s = (Q_r + F_r)} \quad (r, s = 1, \dots, N).$$

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- $M_{rs} = \int_{\mathcal{B}} \varrho c \phi_s \phi_r$ – the thermal capacity matrix $\left[\frac{\text{J}}{\text{K}} \right]$,
- $K_{rs} = \int_{\mathcal{B}} \left(-k \phi_{s|i} + \varrho c u_i \phi_s \right) \phi_{r|i}$ – the heat transfer matrix $\left[\frac{\text{W}}{\text{K}} \right]$,
- $Q_r = \int_{\partial \mathcal{B}_q} \hat{q} \phi_r$ – the inward heat flow vector $[\text{W}]$,
- $F_r = \int_{\mathcal{B}} f \phi_r$ – the heat production vector $[\text{W}]$.

Stationary heat transfer (algebraic equations)

$$T = T(\mathbf{x}) \quad f = f(\mathbf{x}) \quad u_i = u_i(\mathbf{x}) \quad (\text{for } \mathbf{x} \in \mathcal{B})$$

BVP of stationary heat flow: Find $T = T(\mathbf{x})$ satisfying (in \mathcal{B})

$$q_i|_i - f = 0 \quad \text{where:}$$

$$q_i = q_i(T) = \begin{cases} -k T_{|i} & (\text{no convection}), \\ -k T_{|i} + \rho c u_i T & (\text{with convection}), \end{cases}$$

with boundary conditions:

$$T = \hat{T} \quad \text{on } \partial\mathcal{B}_T \text{ (Dirichlet),} \quad -q_i(T) n_i = \hat{q} \quad \text{on } \partial\mathcal{B}_q \text{ (Neumann).}$$

- The weak variational form lacks the rate integrand

$$- \int_{\mathcal{B}} q_i \delta T_{|i} - \int_{\partial\mathcal{B}_q} \hat{q} \delta T - \int_{\mathcal{B}} f \delta T = 0.$$

- The approximations $T(\mathbf{x}) = \theta_s \phi_s(\mathbf{x})$, $\delta T(\mathbf{x}) = \delta\theta_r \phi_r(\mathbf{x})$ lead to the following **system of linear algebraic equations** (for $\theta_s = ?$):

$$\boxed{-K_{rs} \theta_s = (Q_r + F_r)} \quad (r, s = 1, \dots, N).$$